REMARKS

The specification is amended to accord to US review and practice procedures.

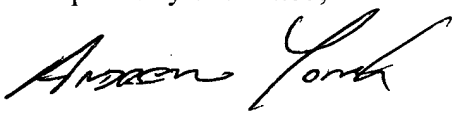
The clean version without markings is present above as required. The marked-up version of the substitute specification excluding claims and with markings to show changes made is provided below as required under §1.121 and §1.125.

No other changes have been made.

No new matter is added.

The Commissioner is hereby authorized to charge payment of any fees associated with this communication, or credit any overpayment, to Deposit Account No. 13-4550.

Respectfully submitted,



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Attached: **VERSION WITH MARKINGS TO SHOW CHANGES MADE**

VERSION WITH MARKINGS TO SHOW CHANGES MADE

IN THE SPECIFICATION:

Kindly amend the specification as follows

CHUCK DEVICE

BACKGROUND OF THE INVENTION

1. Field of the Invention

The present invention relates to a chuck device. More specifically, the present invention relates to a chuck device with a plurality of gear mechanisms which increase a rotational drive force applied through an input member.

2. Description of the Related Art

Conventionally, in machine tools, a chuck device secures a work piece or tool to a work surface. Work surfaces may include a table, a work pallet, or a principal axis clamp. Such chucking devices typically include a base member, secured to the work surface, and a claw member movably mounted on the base member.

Conventional claw members are movable to allow the work piece or tool to be 'chucked' or secured in the chuck device. Chuck devices may include one, two, or three claw members.

Referring now to Fig. 9, a chuck device 100, secures a work piece Wa. Chuck device 100 includes a base member 101. An input shaft member 103 extends from inside base member 101 to project from a side opposite a claw member 102. Chuck device 100 also includes a conversion mechanism 104 and a hydraulic cylinder (not shown).

A leg 102a, of claw member 102, slidably engages a T-shaped groove 101a formed on base member 101. An outer end of input shaft 103 connects to the hydraulic cylinder (not shown).

Conversion mechanism 104 includes a conversion member 105 secured to input shaft member 103. Conversion member 105 includes a sloped engagement groove 105a thereon. Sloped engagement groove 105a has a T-shaped cross-section and is sloped relative to the direction of motion of claw member 102. An engagement section 102b on claw member 102 slidably engages sloped engagement groove 105a.

During operation, the hydraulic cylinder (not shown) drives input shaft member 103 and conversion member 105 in an axial direction. The resulting axial drive force is redirected by conversion mechanism 104. After redirection, the axial drive force is transferred to claw member 102, causing claw member 102 to move in the direction of an arrow a.

Referring now to Fig. 10, a chuck device 110 implemented by the present applicants includes a base member 111, a claw member 112, and an input member 113. Chuck device 110 also includes a conversion mechanism 114.

A leg 112a, on claw member 112, slidably engages a T-shaped groove 111a formed on base member 111. Input member 113, formed as a bolt, is screwed into base member 111. During operation, a rotational drive force is

manually applied to input member 113, using a handle or other manual rotation tool 119, to tighten or loosen chuck device 110.

Conversion mechanism 114 includes a conversion member 115 which receives and engages a head of a shaft of input member 113. A sloped surface 115a on conversion member 115 is sloped relative to a direction of movement of claw member 112. A sloped surface 112b on claw member 112 is in planar contact with sloped surface 115a.

A compression spring 116 elastically biases claw member 112 toward input member 113.

During operation, when input member 113 is rotated in a tightening direction, conversion member 115 is driven downward into base member 111 to force claw member 112 in the direction of an arrow b, thus securing a work piece Wb. When input member 113 is rotated in a loosening direction, the biasing force of compression spring 116 urges claw member 112 to move in the releasing direction of an arrow c to release work piece Wb.

In conventional chuck devices, drive force applied through an input member is marginally increased (multiplied) to drive claw members. Unfortunately, any increase in drive force applied through the input member is limited by the sloped engagement grooves and sloped surfaces used in a conversion mechanism. This is a physical and design limitation which makes it difficult to provide a high force (since there is a lack of a multiplication rate) to increase the ratio to grip a work piece. As a result, in manually driven chuck devices, it is difficult to chuck a work piece or tool firmly. Failure to firmly chuck a work piece or tool may lead to reduced machining precision and damage to cutting tools. Manual operation may result in reduced ease of use and lower production efficiency. Repetitive manual chucking may lead to physically

fatigued operators thus increasing safety risks and extending chucking time. In sum, manual chucking operations reduce productivity.

Unfortunately, where an automatic chuck devices drives the input member, the actuator makes the chuck device larger. The increase in size, increases production costs, production risks, and reduces productivity.

Increasing the slopes of the sloped engagement groove can improve the rate at which the drive force is increased. Unfortunately, the ratio of the displacement of a claw member to a displacement of the conversion member is very small. This ratio limits the size of the work piece or tool that can be chucked, thus further reducing operational versatility.

OBJECTS AND SUMMARY OF THE INVENTION

An object of the present invention is to provide a chuck device that improves and increases a rate of applied drive force.

Another object of the present invention is to provide a chuck device that improves usability and increases the efficiency and force of chucking operations.

Another object of the present invention is to provide a chuck device that is compact.

Another object of the present invention is to provide a highly versatile chuck device, easily adaptable to multiple production environments.

It is another object of the present invention to provide a chuck device that is readily adaptable to one or two claw embodiments, stationary or mobile embodiments, and flat, tilted, or multi-axial positions.

The present invention relates to a chuck device including a first worm gear mechanism linked to a second worm wheel mechanism which operate in

tandem to receive, increase, and redirect an input rotational drive force. A conversion mechanism receives and further augments the drive force from the second worm gear mechanism and converts the drive force into an axial force. The conversion mechanism transfers the axial force symmetrically to a pair of claw members. The claw members move relative to each other and firmly secure a work item to the chuck device.

According to an embodiment of the present invention, there is provided a chuck device comprising: a first base member, a second base member on the first base member, first means for receiving and increasing a rotational force, the first means for receiving and increasing in the first base member, second means for receiving the rotational force from the first means and for further increasing the rotational force into an increased rotational force, the second means for receiving in the first base member, the second means for receiving effective to redirect the increased rotational force perpendicular to the first means for receiving and increasing, means for converting the increased rotational force from the second means into an increased axial force perpendicular to the first and the second means, and the means for converting operable between the first and the second base member, whereby the rotational force is transferred through the first base member to the second base member and converted into an increased axial force operable relative to the second base member.

According to another embodiment of the present invention there is provided a chuck device, further comprising: means for chucking an external item in the second base member, and the means for chucking receiving the increased axial force and securely chucking the external item to the second base member, whereby the external item is easily secured with a holding force magnified from the rotational force.

According to another embodiment of the present invention there is provided a chuck device, further comprising: at least a first conversion member in the means for converting, the second means for receiving effective to drive the first conversion member away from the first means for receiving and increasing, at least a first sloped engagement groove on the first conversion member, at least a first claw member in the means for chucking, at least a first engagement section on the first claw member, the first sloped engagement groove sloped relative to a first direction of motion of the first claw member relative to the second base member, the means for chucking effective to operate the at least first claw member axially along an axial direction of the second base member, and the first sloped engagement groove engaging the first engagement section effective to retain the first engagement section and to drive the first engagement section in the first direction of motion and fix the external item to the second base member, whereby the external item is secured to the chuck device.

According to another embodiment of the present invention there is provided a chuck device, further comprising: at least a first worm gear in the first means for receiving and increasing, at least a second worm wheel in the second means for receiving and for further increasing, the rotational force operable about a first diameter, the first worm gear having a first rotational axis and a second diameter, the second diameter greater than the first diameter, the second worm wheel having a second rotation axis, the first rotational axis perpendicular to the second rotational axis, and the first worm gear threadably engaging the second worm wheel and effective to magnifying the rotational force.

According to another embodiment of the present invention there is provided a chuck device, further comprising: a first operational axis on the means for converting, the first operation axis parallel the second rotational axis, the first

operational axis perpendicular the an axial direction of motion of the first claw member, the first operational axis perpendicular to the first rotational axis, the first sloped engagement groove sloped relative to first operation axis, and the means for converting effective to receive the increased rotational force and operate along the second rotational axis, whereby the first claw member operates simultaneously in the first direction of motion relative to the second base member and the along the first sloped engagement groove relative to the first conversion member.

According to another embodiment of the present invention there is provided a chuck device, further comprising: at least a first engagement groove in the second base member, at least a first leg on first claw member, the first leg in the first engagement groove, and the first engagement groove effective to engage the first leg and operate the first claw member axially along the first direction of motion.

According to another embodiment of the present invention there is provided a chuck device, wherein: the first sloped engagement groove has a slope on or about 70 degrees relative to a direction of motion of the first claw member.

According to another embodiment of the present invention there is provided a chuck device, wherein: the first base member includes at least a first hole and a second hole, the first worm gear in the first hole, the second worm wheel in the second hole, at least a first cover, the first cover on at least a first face of the first base member, the at least first cover effective to operably retain the first worm gear in the first hole and allow external input of the rotational force, at least a second cover, the second cover on a second face of the first base member opposite, the first face perpendicular to the second face, and the at least second

cover effective to operably retain the second worm wheel in the second hole and allow operation of the conversion member relative to the worm wheel.

According to another embodiment of the present invention there is provided a chuck device, further comprising: at least a first grease access in the at least first claw member, the first grease access parallel the first direction of motion, and the first grease access operable along a first face of the first sloped engagement groove, whereby an external lubricant is easily applied between the conversion member and the first engagement section effective to allow smooth operation of the chuck device.

According to another embodiment of the present invention there is provided a chuck device, further comprising: a second sloped engagement groove on the first conversion member, a second claw member in the means for chucking, at least a second engagement section on the second claw member, the second sloped engagement groove sloped relative to a second direction of motion of the second claw member relative to the second base member, the means for chucking effective to operate the second claw member axially along the axial direction of the second base member, and the second sloped engagement groove engaging the second engagement section effective to retain the second engagement section and drive the second engagement section along the second direction of motion and fix the external item to the second base member, whereby the external item is secured to the chuck device.

According to another embodiment of the present invention there is provided a chuck device, comprising: a first base member, a second base member on the first base member, first means for receiving and increasing a rotational force, the first means for receiving and increasing in the first base member, second means for receiving the rotational force from the first means and for further

increasing the rotational force into an increased rotational force, the second means for receiving in the first base member, the second means for receiving effective to redirect the increased rotational force perpendicular to the first means for receiving and increasing, means for converting the increased rotational force from the second means into an increased axial force perpendicular the first and the second means, the means for converting operable between the first and the second base member, whereby the rotational force is transferred through the first base member and into the second base member and converted into an increased axial force operable relative to the second base member, means for chucking an external item in the second base member, the means for chucking receiving the increased axial force and securely chucking the external item to the second base member, whereby the external item is easily secured with a holding force magnified from the rotational force, at least a first conversion member in the means for converting, the second means for receiving effective to drive the first conversion member away from the first means for receiving and increasing, at least a first sloped engagement groove on the first conversion member, at least a first claw member in the means for chucking, at least a first engagement section on the first claw member, the first sloped engagement groove sloped relative to a first direction of motion of the first claw member relative to the second base member, the means for chucking effective to operate the at least first claw member axially along an axial direction of the second base member, and the first sloped engagement groove engaging the first engagement section effective to retain the first engagement section and to drive the first engagement section in the first direction of motion and fix the external item to the second base member, whereby the external item is secured to the chuck device.

According to another embodiment of the present invention there is provided a chuck device, comprising: first means for receiving and increasing a rotational force, second means for receiving and increasing the rotational force from the first means and outputting an increased rotational force, the second means for receiving redirecting and rotational force from a first base member to a second base member, means for receiving and converting the increased rotational force from the second means into an increased axial force, means for chucking an external item to the second base member, and the means for chucking receiving the increased axial force and securing the external item to the second base member and the chuck device.

According to another embodiment of the present invention there is provided a chuck device, further comprising: at least a first conversion member in the means for receiving and converting, at least a first sloped engagement groove on the first conversion member, at least a first claw member in the means for chucking, at least a first engagement section on the first claw member, the first sloped engagement groove sloped relative to a direction of motion of the first claw member, the means for chucking effective to operate the at least first claw member axially along an axial direction of the second base member, and the first sloped engagement groove engaging the first engagement section effective to drive the first engagement section in the direction of motion and fix the work item in the second base member, whereby the work item is secured in the chuck device.

According to another embodiment of the present invention there is provided a chuck device, further comprising: at least a first worm gear in the first means for receiving and increasing, at least a second worm wheel in the second means for receiving and increasing, the first worm gear having a first rotational axis, the second worm wheel having a second rotation axis, the first rotational axis

perpendicular to the second rotational axis, and the first worm gear operably threadably engaging the second worm and magnifying the rotational force.

According to another embodiment of the present invention there is provided a chuck device, including a first base member and at least a first claw member movably mounted on the first base member, for chucking an external item by moving the first claw member, the chuck device comprising: a first input member for receiving and applying a rotational force, a first gear mechanism effective to receive and increase the rotational force, a second gear mechanism effective to receive the rotational force from the first gear mechanism, increase the rotational force, and operate a screw shaft member along an axial direction perpendicular to the first input member, and a conversion mechanism effective to receive the rotational force, resist rotation relative to the first input member, and convert the rotational force into an axial force to drive the at least first claw member in an axial direction relative the first base member, whereby the chuck device securely engages the external item.

According to another embodiment of the present invention there is provided a chuck device, further comprising: a worm gear in the first gear mechanism, a worm wheel in the second gear mechanism, the worm gear rotating integrally with the input member, the worm gear threadably engaging and the worm wheel, the second gear mechanism including a threaded hole concentric with a center of the worm wheel, and a screw shaft member threadably engaging the threaded hole.

According to another embodiment of the present invention there is provided a chuck device, wherein: the conversion mechanism includes a conversion member, the conversion member secured to the screw shaft member and at least a first sloped engagement groove on the conversion member, the first

sloped engagement groove sloped relative to a direction of motion of the first claw member, at least a first engagement section on the first claw member, and the first engagement section slidably engaging the first sloped engagement groove and preventing the conversion member from rotating relative to the worm wheel.

According to another embodiment of the present invention there is provided a chuck device, further comprising: at least the first and a second claw member, the first and the second claw members disposed facing each other on the base member, a first leg on the first claw member, a second leg on the second claw member, the first and the second legs slidably engaging a shared engagement groove on the base member effective to axially align the first and the second claw members, at least a second sloped engagement groove on the conversion member, the second sloped engagement groove sloped relative to a direction of motion of the second claw member, at least a second engagement section on the second claw member, the second engagement section slidably engaging the second sloped engagement groove and preventing the conversion member from rotating relative to the worm wheel, and the conversion mechanism effective to slidably engage and move the first and the second claw member symmetrically along the shared engagement groove.

The present invention provides a chuck device for chucking a workpiece or a tool by moving single or multiple claw members. The chuck device includes a base member and at least one claw member movably mounted on the base member. The chuck device also includes an input member for applying a rotational drive force, a gear mechanism using a rotational drive force applied through the input member to drive a screw shaft member in an axial direction, and a conversion mechanism redirecting an axial drive force transferred through the screw shaft member and driving a claw member.

The above, and other objects, features and advantages of the present invention will become apparent from the following description read in conjunction with the accompanying drawings, in which like reference numerals designate the same elements.

BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1 is a perspective drawing of a chuck according to an embodiment of the present invention.

Fig. 2 is a plan drawing of the chuck device of Fig. 1.

Fig. 3 is a front-view drawing of the chuck device of Fig. 1.

Fig. 4 is a vertical cross-section drawing of the chuck device in a chucked state.

Fig. 5 is a vertical cross-section drawing of the chuck device in an unchucked state.

Fig. 6 is a cross-section drawing taken along the VI-VI line in Fig. 4.

Fig. 7 is a vertical cross-section drawing of a conversion member.

Fig. 8 is a plan drawing of a conversion member.

Fig. 9 is a vertical cross-section of a conventional chuck device.

Fig. 10 is a vertical cross-section drawing of another conventional chuck device.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

Referring now to Figs. 1 and 2, a two-claw-type chuck device 1 includes a base member 2 and a pair of claw members 3. Chuck device 1 also includes

front and rear input shaft members 4 and a worm gear mechanism 5 (shown later) in base member 2. A second gear mechanism 6 (shown later), and a conversion mechanism 7 extend from base member 2 into an upper block 10. Claw members 3 fix a workpiece W in chuck device 1, as will be described.

During operation, input shaft member 4 receives an input rotational force, not shown. Input shaft member 4 rotates to transmit an input rotational force to worm gear mechanism 5, second gear mechanism 6, and conversion mechanism 7. The input rotational force is magnified and transferred to claw members 3. During operation, a left claw member 3b and a right claw member 3c move symmetrically to axially inward to fix workpiece W in chuck device 1, as will be explained.

The illustrated base member 2 is a wide rectangular shape when seen from above. However, base member 2 may be of any shape sufficient to embody the present invention and support workpiece W.

Base member 2 includes integrally formed upper block 10 and a lower block 20. Lower block 20 is wider than upper block 10 for stability, but may have other shapes sufficient to stabilize workpiece W.

Claw members 3 are movably mounted on an upper surface (upper surface section) of upper block 10. Four bolt holes 2a are located in the corner areas of lower block 20. Four bolts (not shown) are insertable into bolt holes 2a to secure base member 2 to the table (not shown) of a machine tool (not shown) to support lower block 20.

An engagement groove 11 along a left-right axis of upper block 10 has an approximately T-shape cross-section.

A pair of legs 30 (only one of which is shown), on respective claw members 3, are slidably engaged in shared engagement groove 11. Upper block

10 includes a lowered section 12 to form a shelf 12a at a central section along the left-right axis. A hole 13 in upper block 10 extends downward from lowered section 12 to lower block 20 to communicate with engagement groove 11.

Claw members 3 are symmetrically positioned in upper block 10. Each claw member 3 includes a leg 30 with a main claw unit 31 extending above leg 30, terminating in an engagement section 52. Engagement sections 52 are located on the end of each leg 30, near lowered section 12.

Conversion mechanism 7 includes a conversion member 50, a sloped engagement grooves 51, and engagement sections 52.

Main claw unit 31 includes a pair of front and rear claw sections 31a extending upward from the upper end of leg 30. A groove 31b is formed inside main claw unit 3 and is surrounded by claw sections 31a and the upper end of leg 30.

Front and rear claw sections 31a extend parallel to each other along the left-right axis of chuck device 1. The upper ends of front and rear claw sections 31a of the left and right claw members 3 face each other. The facing ends of left and right claw sections 31a chuck (hold) workpiece W on lowered section 12 thereby supporting workpiece W from both ends.

A horizontal hole 21 on a first side of lower block 20 extends through from the first side to the second side of lower block 20. A pair of covers 24 are fitted into a front and a rear side of horizontal hole 21 (rear side cover 24 not shown). During operation, covers 24 prevent input shaft member 4 from slipping out of lower block 20.

An angular hole 4a at an outer end of input shaft member 4 receives an end of a rotation tool, such as a hexagonal wrench. Rotation of the rotation tool,

rotates input shaft member 4 to apply rotational drive force to tighten or loosen chuck device 1.

Referring now to Figs. 3 to 5, a vertical hole 22 extends upward from a bottom of lower block 20 to a center portion of lower block 20. A right end of vertical hole 22 communicates with horizontal hole 21, as will be described. A vertical hole 23 extends upward from vertical hole 22 into upper block 10. Vertical hole 23 communicates with hole 13. A cover 25 is fitted into the bottom of vertical hole 22, as will be described.

During operation, input shaft members 4 receive input rotational drive force for application to chuck device 1. Front and rear input shaft members 4 in horizontal hole 21 of base member 2 is fitted into lower block 20 where they are rotatably supported by respective covers 24.

Worm gear mechanism 5 in horizontal hole 21 includes a worm gear 40. During assembly, inner ends of input shaft members 4 are inserted into worm gear 40. A key member 26, links input shaft members 4 to worm gear 40 to prevent their relative rotation.

Each leg 30 of each claw member 3 include a grease hole 3a. During operation and maintenance, lubricant fed into grease holes 3a and between sloped engagement grooves 51 and engagement sections 52 lubricates chuck device 1.

Worm gear mechanism 5, mounted inside base member 2, includes a worm gear 40, which rotates integrally with input shaft members 4 in mesh with a worm wheel 41.

Worm wheel 41 is disposed within vertical hole 22 of base member 2. Cover 25 and base member 2 rotatably support worm wheel 41 to prevent worm wheel 41 from moving along its axis. Worm wheel 41 is rotatably fitted and screwed to a screw shaft member 46 of second gear mechanism 6.

Second gear mechanism 6 is mounted in base member 2. Second gear mechanism 6 drives screw shaft member 46 axially using the rotational drive force transferred from worm gear mechanism 5.

Second gear mechanism 6 includes a threaded hole 45 formed concentric wit a center of worm wheel 41. During assembly, screw shaft member 46 is screwed into threaded hole 45. A bolt 47 is inserted in the center of screw shaft member 46 with the threaded section of bolt 47 projecting upward beyond screw shaft member 46 to link into conversion member 50 of conversion mechanism 7 so that screw shaft member 46 and conversion member 50 are fixed together.

A collar member 48 is secured between screw shaft member 46 and a head of bolt 47. Collar member 48 has a diameter that is slightly smaller than that of a hole 25a of cover 25.

During operation, when screw shaft member 46 is in a lowered state, collar member 48 fits into hole 25a to place the axial center of the screw shaft member 46 and the rotational center of the worm wheel 41 in fixed positions.

Conversion mechanism 7 changes the direction of the axial drive force transferred by screw shaft member 46, thereby transferring this force symmetrically to left-right force on claw members 3.

Referring now to Figs. 6 to 8, worm gear mechanism 5 increases drive torque by slowing down the rotational drive force input from input shaft member 4.

Conversion member 50 is secured to screw shaft member 46 by bolt 47. Conversion member 50 is disposed within holes 13 and vertical hole 23 in base member 2. Conversion member 50 includes T-shaped sloped engagement grooves 51 each sloped in the direction of movement of its respective claw member 3.

Claw members 3 each include engagement sections 52 which slidably engage sloped engagement grooves 51. Each engagement section 52 engages its respective sloped engagement groove 51, thereby preventing conversion member 50 from rotating relative to base member 2. Since each engagement section 52 engages with each sloped engagement groove 51 on conversion member 50, claw members 3 operate simultaneously to capture or release workpiece W.

In one particular embodiment of the present invention, sloped engagement grooves 51 are sloped at approximately 70 degrees relative to the direction of motion of claw members 3 (the horizontal direction). During operation, as engagement grooves 50 move lower relative to a top surface of upper block 10, they move further from the axial center of conversion member 50.

As the conversion member 50 and screw shaft member 46 move lower relative to the top surface of upper block 10, claw members 3 move closer to each other. Conversely, as conversion member 50 and screw shaft member 46 move higher, the claw members 3 move further apart.

The operations and advantages of the two-claw chuck device 1 will be described.

During operation, when chucking workpiece W into chuck device 1, conversion member 50 is first moved upward to provide adequate space between claw members 3. Workpiece W is then set onto lowered section 12. When conversion member 50 is moved to an uppermost position, claw members 3 are separated by a maximum distance. However, when setting workpiece W, claw members 3 may not need to be separated by the maximum distance and are adjustable according to operational needs.

After insertion, the end of the rotation tool (hexagonal wrench) engages angular hole 4a of input shaft member 4 to manually rotate input shaft member 4

thereby to move claw members 3 toward each other. In worm gear mechanism 5, worm gear 40 rotates integrally with input shaft member 4, and worm wheel 41, meshed with worm gear 40, rotates around the vertical axis.

In second gear mechanism 6, worm wheel 41 is restrained from vertical movement when worm wheel 41 is rotated, and screw shaft member 46, screwed into threaded hole 45 of worm wheel 41 is driven downward. Conversion member 50, secured to screw shaft member 46 is correspondingly lowered.

In conversion mechanism 7, when conversion member 50 moves downward, sloped engagement grooves 51, and engagement sections 52 cause claw members 3 to move toward workpiece W.

Upon tightening, the upper ends of claw sections 31a of claw members 3 firmly chuck workpiece W, thereby securing workpiece W for machining. Machining is then performed on workpiece W in a chucked state.

When removing workpiece W, the rotation tool rotates input shaft member 4 in an opposite direction to separate claw members 3. The rotation causes screw shaft member 46 to move upward, thus causing conversion member 50, secured to screw shaft member 46, to move upward as well, thereby driving claw members 3 apart (through conversion mechanism 7) to releases workpiece W.

According to the present invention, worm gear mechanism 5 allows the rotational drive force input through input shaft member 4 to be significantly mechanically multiplied. Furthermore, second gear mechanism 6 further increases the rotational drive force transferred from worm gear mechanism 5 before transferring the force to screw shaft member 46 and driving screw shaft member 46 in the axial direction. The axial drive force transferred by screw shaft member 46 is redirected and increased by conversion mechanism 7 and is

transferred substantially equally to claw members 3. Claw members 3 move symmetrically.

The drive force input through input shaft member 4 can be increased in three stages and then transferred to claw members 3. As a result, simply applying a manual drive torque to input shaft member 4 can easily and firmly chuck workpiece W into chuck device 1. This improves the usability of chuck device 1 to make chucking operations more efficient, thus minimizing machining imprecision and damage to cutting tools.

A high ratio for increasing the drive force can be provided even without a very large ratio between the displacement stroke of claw members 3 and the displacement strokes of conversion member 50 and screw shaft member 46.

Since claw members 3 are mounted on the upper surface of base member 2, and since worm gear mechanism 5 and second gear mechanism 6 are mounted inside base member 2, the structure for increasing the drive force is compact. The entry of debris, such as cuttings, into gear mechanisms 5, 6 is minimized.

Since claw members 3 face each other, and since legs 30 slidably engage shared engagement groove 11, shared engagement groove 11 reliably guides and supports claw members 3 on a common axis. As a result, workpiece W is reliably chucked and supported from either side by claw members 3.

Multiple additional embodiments of chuck device 1 are described below each containing the essence of the invention.

In a another embodiment, worm gear mechanism 5 may be omitted and a rotation member may be provided to substitute for worm wheel 41. This allows screw shaft member 46 to screw into a threaded hole formed in the rotation member. Alternative input members to receive rotational drive force may be provided to rotate the rotation member and drive screw shaft member 46 axially.

In another embodiment, a single claw member may replace claw members 3. In this embodiment, workpiece W is chucked by supporting it between the claw member and a receiving section of base member 2. Alternatively, three claw members can be provided. Workpiece W can be chucked using these three claw members. Changes in the number of claw members can be easily handled simply by changing the number and positions of the sloped engagement grooves formed on conversion member 50 in conversion mechanism 7.

In a third embodiment, input shaft member 4 may be rotated by an actuator such as a motor to apply rotational drive force to input shaft member 4. Since rotation of input shaft member 4 does not require a high drive force, the actuator can be compact, thus allowing chuck device 1 to be compact and minimize production costs.

In a fourth embodiment, an upper plate, used to chuck workpiece W by supporting it in cooperation with one of claw members 3, can be secured to lowered section 12 using bolts screwed into lowered section 12. Here, the other claw member 3 is not used. In this embodiment, mounting the upper plate adapted to work pieces having unusual shapes, makes it possible to chuck or secure work pieces smaller than work piece W. Furthermore, after the upper plate is mounted, the upper plate may adapt to a particular shape corresponding to the shape of an oddly shaped work piece. This embodiment provides reliable chucking for different or oddly shaped work pieces W.

The present invention may be used in chuck devices that secure the work piece to a rotating body of a machine tool or that secure tools to a principal operational axis.

With chuck device 1, according to the present invention, a light manual drive force, applied to drive claw members 3, is multiplied to forcefully and

firmly chuck workpiece W, thus increasing operation efficiency. As described above, 'chucking' is meant to indicate securely retaining a work piece or tool in chuck device 1. During operation of chuck device 1, either a work piece or a tool should be understood as a work item. The phrase 'work item' represents an item which must be held securely and steadily in chuck device 1 and may be either a work piece or a tool used on a work piece.

Chuck device 1 may include from one-claw to multi-claw embodiments adapted from the principal of the present invention. For example, in a one-claw embodiment, one of the claws is replaced with a fixed member. In a three-claw embodiment, the claws operate radially from a common center to fix a workpiece.

It should be understood, that chuck device 1 operates as a way to chuck or retain items, such as work piece W or a tool, during a processing operation.

During operation, when rotational drive force is applied through the input member, the rotational drive force drives the screw shaft member in the axial direction via a gear mechanism. The axial drive force transferred to the screw shaft member is redirected by a conversion mechanism for transfer to the claw member, thereby moving the claw member. The gear mechanism allows the rotational drive force applied through the input member to be significantly increased as it is transferred to the screw shaft member.

Since the drive force applied to the input member is multiplied significantly as it is transferred to the claw member, a work piece or tool can be firmly chucked by the claw member by applying a relatively small drive force to the input member. This ability to multiply input force improves the efficiency of chucking operations.

During operation, if an actuator is used to drive the input member, a relatively small actuator can be used, thus allowing the chuck device to be compact and reducing production costs.

A high drive rate increase ratio can be provided without excessively increasing the ratio of the displacement stroke of the screw shaft member or the like to the displacement stroke of the claw member.

In the chuck device described, the gear mechanism may include a worm gear mechanism for slowing down the rotational drive force applied through the input member and a second gear mechanism that uses the rotational drive force transferred from the worm gear mechanism to drive the screw shaft member in an axial direction. The worm gear mechanism significantly increases the rotational drive force applied to the input member and provided a great benefit. The rotational drive force increased by the worm gear mechanism is further increased by the second gear mechanism, which transfers the drive force to the screw shaft member, driving it in the axial direction. Operating together, an initial rotational force is greatly increased to permit the chuck mechanism to be easily, simply, and accurately adjusted.

During operation, when screw shaft member 46 is driven in the axial direction, conversion mechanism 7 drives conversion member 50 integrally with screw shaft member 46, changing the engagement position of engagement sections 52 in sloped engagement grooves 51. As a result, the drive force in the axial direction from screw shaft member 46 is further multiplied when transferred to claw member 3.

Although chuck device 1 is shown with a pair of claw members 3, a single claw member can be used on chuck device 1 to retain work piece W between the single claw member and a fixed member or an external fixed or movable member.

The input member of the chuck device can be driven manually but is easily adapted for driving by a small electrical or hydraulic actuator.

Although only a single or few exemplary embodiments of this invention have been described in detail above, those skilled in the art will readily appreciate that many modifications are possible in the exemplary embodiment(s) without materially departing from the novel teachings and advantages of this invention. Accordingly, all such modifications are intended to be included within the spirit and scope of this invention as defined in the following claims. In the claims, means-plus-function clauses are intended to cover the structures described or suggested herein as performing the recited function and not only structural equivalents but also equivalent structures. Thus, for example, although a nail, a screw, and a bolt may not be structural equivalents in that a nail relies entirely on friction between a wooden part and a cylindrical surface, a screw's helical surface positively engages the wooden part, and a bolt's head and nut compress opposite sides of the wooden part together, in the environment of fastening wooden parts, a nail, a screw, and a bolt may be readily understood by those skilled in the art as equivalent structures.

Having described preferred embodiments of the invention with reference to the accompanying drawings, it is to be understood that the invention is not limited to those precise embodiments, and that various changes and modifications may be effected therein by one skilled in the art without departing from the scope or spirit of the invention as defined in the appended claims.

ABSTRACT OF THE DISCLOSURE

A chuck device includes a first worm gear mechanism linked to a second worm wheel mechanism which operate in tandem to receive, increase, and redirect

an input rotational drive force. A conversion mechanism receives and further augments the drive force from the second worm gear mechanism and converts the drive force into an axial force. The conversion mechanism transfers the axial force symmetrically to a pair of claw members. The claw members move relative to each other and firmly secure a work item to the chuck device.

~~Specification~~

~~Chuck device~~

~~Technical field~~

~~The present invention relates to a chuck device. More specifically, the present invention relates to a chuck device equipped with a gear mechanism that increases rotational drive force applied through an input member.~~

~~Background technology~~

~~Conventionally, in machine tools such as milling machines, lathes, and machining centers, a chuck device is used to secure a workpiece to a table, work pallet, or the like, or to mount a tool to the principle axis, or the like. A chucking device essentially includes a base member secured to a table, a work pallet, a primary axis, or the like, and a claw member movably mounted on this base member. This claw member is moved so that the workpiece or tool is chucked. Conventionally, one-claw chuck devices with a single claw member, two-claw chuck devices with two claw members, and three-claw chuck devices with three claw members have been used practically.~~

Referring to Fig. 9, for example, a chuck device 100, used to secure a workpiece Wa, includes a base member 101, a claw member 102, an input shaft member 103, a conversion mechanism 104, and a hydraulic cylinder (not shown in the figure). A leg 102a of the claw member 102 slidably engages with a T-shaped groove 101a formed on the base member 101. The input shaft member 103 extends from inside the base member 101 and projects from the side opposite from the claw member 102. The outer end is connected to the hydraulic cylinder.

The conversion mechanism 104 includes: a conversion member 105 secured to the input shaft member 103; a sloped engagement groove 105a formed on the conversion member 105 with a T-shaped cross-section shape and sloped relative to the direction of motion of the claw member 102; and an engagement section 102b disposed on the claw member 102 to slidably engage with the sloped engagement groove 105a. The hydraulic cylinder drives the input shaft member 103 and the conversion member 105 in the axial direction, and this axial drive force is redirected by the conversion mechanism 104 and transferred to the claw member 102, causing the claw member 102 to move in the direction of the arrow a.

Referring to Fig. 10, a chuck device 110 implemented by the present applicants includes a base member 111, a claw member 112, an input member 113, and a conversion mechanism 114. A leg 112a of the claw member 112 slidably engages with a T-shaped groove 111a formed on the base member 111. The input member 113, formed as a bolt, is screwed into the base member 113, and a rotational drive force is applied manually to the input member 113 using a rotation tool 119.

The conversion mechanism 114 includes: a conversion member 115 into which the shaft of the input member 113 is inserted and which engages with the

head of the input member 113; a sloped surface 115a formed on the conversion member 115 and sloped relative to the direction of movement of the claw member 112; a sloped surface 112b formed on the claw member 112 to form a planar contact with the sloped surface 115a; and a compression spring 116 elastically biasing the claw member 112 toward the input member 113. If the input member 113 is rotated in the tightening direction, the conversion member 115 is driven downward, and the claw member 112 is moved in the direction of the arrow b via the conversion mechanism 114, thus securing the workpiece W. If the input member 113 is rotated in the loosening direction, the biasing force of the compression spring 116 causes the claw member 112 to move in the direction of the arrow c.

In conventional chuck devices, the drive force applied through the input member can be increased (multiplied) by a conversion mechanism to drive a claw member. However, the increase in drive force applied through the input member is limited if only sloped engagement grooves and sloped surfaces are used in the conversion mechanism for increasing drive force. This makes it difficult to provide a high force increase ratio (multiplication rate).

As a result, with chuck devices in which an input member is manually driven, the workpiece or tool cannot be firmly chucked. This can lead to reduced machining precision and damage to cutting tools. Thus, firmly driving the input member manually results in reduced ease of use and lowers the efficiency of the chucking operation. Also, repeating this chucking operation will lead to fatigued arms and hands. With chuck devices that drive the input member using an actuator such as a hydraulic cylinder, the actuator makes the chuck device larger and increases production costs.

~~However, increasing the slopes of the sloped engagement groove and the sloped surface of the conversion member in the conversion mechanism can improve the rate at which the drive force is increased. However, the ratio of the displacement of the claw member to the displacement of the conversion member becomes very small. This limits the size of the workpiece, tool, or the like that can be chucked, thus reducing its versatility.~~

~~The object of the present invention is to provide a chuck device that can improve the increase rate of the applied drive force. Another object of the present invention is to provide a chuck device that improves usability and increases the efficiency of chucking operations. Yet another object of the present invention is to provide a chuck device that can be designed compactly. Yet another object of the present invention is to provide a highly versatile chuck device.~~

~~Disclosure of the invention~~

~~The present invention provides a chuck device for chucking a workpiece or a tool by moving a claw member. The chuck device includes a base member and at least one claw member movably mounted on the base member. The chuck device also includes: an input member for applying a rotational drive force; a gear mechanism using a rotational drive force applied through the input member to drive a screw shaft member in an axial direction; and a conversion mechanism redirecting an axial drive force transferred through the screw shaft member and driving a claw member.~~

~~When rotational drive force is applied through the input member, the rotational drive force drives the screw shaft member in the axial direction via a gear mechanism. The axial drive force transferred to the screw shaft member is redirected by a conversion mechanism and transferred to the claw member,~~

moving the claw member. The gear mechanism allows the rotational drive force applied through the input member to be significantly increased and transferred to the screw shaft member.

Since the drive force applied to the input member can be multiplied significantly and transferred to the claw member, a workpiece or tool can be firmly chucked by the claw member by applying a relatively small drive force to the input member. This improves the efficiency of chucking operations.

If an actuator is used to drive the input member, a relatively small actuator can be used, thus allowing the chuck device to be compact and reducing production costs.

A high drive rate increase ratio can be provided without excessively increasing the ratio of the displacement stroke of the screw shaft member or the like to the displacement stroke of the claw member.

In the chuck device described above, the gear mechanism can include a worm gear mechanism for slowing down the rotational drive force applied through the input member and a second gear mechanism that uses the rotational drive force transferred from the worm gear mechanism to drive the screw shaft member in an axial direction. In this chuck device, the worm gear mechanism significantly increases the rotational drive force applied to the input member. The rotational drive force increased by the worm gear mechanism is further increased by the second gear mechanism, which transfers the drive force to the screw shaft member, driving it in the axial direction.

It would be desirable for the conversion mechanism to include: a conversion member secured to the screw shaft member and not rotating relative to the base member; a sloped engagement groove formed on the conversion member and sloped relative to a direction of motion of the claw member; and an

engagement section disposed on the claw member and slidably engaging with the sloped engagement groove. When the screw shaft member is driven in the axial direction, the conversion mechanism drives the conversion member integrally with the screw shaft member, changing the engagement position of the engagement section in the sloped engagement groove. As a result, the drive force in the axial direction from the screw shaft member is multiplied and transferred to the claw member.

The chuck device can be formed in the following manner. A pair of claw members are disposed facing each other, legs of the claw members are slidably engaged with a shared engagement groove formed on the base member, and the conversion mechanism is formed to move the pair of claw members symmetrically. Alternatively, a single claw member can be disposed on the chuck device.

The input member of the chuck device can be driven manually. Alternatively, a small electrical or hydraulic actuator can be used to drive the input member.

Brief description of the drawings

Fig. 1 is a perspective drawing of an embodiment of the present invention. Fig. 2 is a plan drawing of a chuck device. Fig. 3 is a front-view drawing of a chuck device. Fig. 4 is a vertical cross-section drawing of a chuck device (in a chucked state). Fig. 5 is a vertical cross-section drawing of a chuck device (in an unchucked state). Fig. 6 is a cross-section drawing along the VI-VI line in Fig. 4. Fig. 7 is a vertical cross-section drawing of a conversion member. Fig. 8 is a plan drawing of a conversion member. Fig. 9 is a vertical cross-section drawing of a

chuck device according to a conventional technology. Fig. 10 is a vertical cross-section drawing of a chuck device according to another conventional technology.

Preferred embodiments of the invention

Referring to the figures, the following is a description of an embodiment of the present invention.

This embodiment is an example of the present invention implemented in a two-claw chuck device equipped with a pair of claw members disposed symmetrically. These claw members are moved to chuck a workpiece, securing it to the table of a machine tool or the like.

Referring to Fig. 1 through Fig. 6, a two-claw chuck device 1 includes a base member 2, left and right claw members 3, input shaft members 4, a worm gear mechanism 5, a second gear mechanism 6, and a conversion mechanism 7. With this chuck device 1, the input shaft member 4 is rotated manually so that the drive force is transferred to the left and right claw members via the worm gear mechanism 5, the second gear mechanism 6, and the conversion mechanism 7, thus moving the claws 3 symmetrically.

Referring to Fig. 1 through Fig. 6, the base member 2 forms a wide rectangular shape when seen from above. The base member 2 includes an upper block 10 and a lower block 20 formed integrally thereto. The lower block 20 is formed slightly wider than the upper block 10. The claw members 3 are movably mounted on the upper surface (upper surface section) of the upper block 10. Four bolt holes 2a are formed at the corner areas of the lower block 20. Four bolts (not shown in the figures) inserted into these bolt holes 2a secure the base member 2 to the table of the machine tool (not shown in the figure) or the like.

An engagement groove 11 having a roughly T-shaped cross-section shape is formed on the upper block 10 extending along the left-right axis. Legs 30 of the pair of claw members 3 are slidably engaged with this shared engagement groove 11. The upper block 10 is formed with a lowered section 12 via a shelf 12a at the central section along the left-right axis. A vertical hole 13 communicating with the engagement groove 11 is formed extending downward from the lowered section 12.

The pair of claw members 3 are formed symmetrically to the left and the right. Each claw member 3 includes: the leg 30; a main claw unit 31 disposed above the leg 30; and an engagement section 52 disposed at the end of the leg 30 toward the lowered section 12. The main claw unit 31 is formed from front and rear claw sections 31a extending upward from the upper end of the leg 30. A groove 31b is formed inside the main claw unit 3, surrounded by the claw sections 31a and the upper end of the leg 30. The front and rear claw sections 3a extend parallel to each other along the left-right axis. The upper ends of the front and rear claw sections 3a of the left and right claw members 3 face each other and the facing ends of the left and right claw sections 31a serve to chuck a workpiece W disposed on the lowered section 12 by supporting it from both ends.

A horizontal hole 21 on the right side of the lower block 20 of the base member 2 extends from the front to the rear. A vertical hole 22 extends from the bottom of the lower block 20 at the central area thereof. The right end of the vertical hole 22 communicates with the horizontal hole 21. A vertical hole 23 is formed upward from the upper end of the vertical hole 22, and the vertical hole 23 communicates with the vertical hole 13. A pair of covers 24 are fitted into the front and rear ends of the horizontal hole 21, and a cover 25 is fitted into the bottom of the vertical hole 22.

~~Referring to Fig. 1 and Fig. 3 through Fig. 6, the input shaft members 4 is a member used to input rotational drive force. The front and rear input shaft members are disposed in the horizontal hole 21 of the base member 2. Each input shaft member 4 is fitted in and rotatably supported by the cover 24. A worm gear 40 of the worm gear mechanism 5 is disposed in the horizontal hole 21. The inner ends of the input shaft members 4 are inserted into the worm gear 40, and the input shaft members 4 are linked to the worm gear 40 via a key member 26 in a manner that prevents rotation relative to each other.~~

~~An angular hole 4a, e.g., a hexagonal hole, is formed at the outer end of the input shaft member 4. The end of a rotation tool such as a hexagonal wrench is engaged with the angular hole 4a, and the input shaft member 4 is manually rotated via this rotation tool to input rotational drive force. The input shaft member 4 is prevented from slipping out by the cover 24.~~

~~Referring to Fig. 4 through Fig. 6, the worm gear mechanism 5 is a mechanism for increasing drive torque by slowing down the rotational drive force input via the input shaft member 4.~~

~~The worm gear mechanism 5 is mounted inside the base member 2. This worm gear mechanism 5 includes the worm gear 40, which rotates integrally with the input shaft members 4, and a worm wheel 41, which meshes with the worm gear 40. The worm wheel 41 is disposed inside the vertical hole 22 of the base member 2, and is rotatably supported by the base member 2 and the cover 25 while being prevented from moving along its axis. The worm wheel 41 is rotatably fitted and screwed to a screw shaft member 46 of the second gear mechanism 6.~~

Referring to Fig. 4 and Fig. 5, the second gear mechanism 6 is a mechanism for driving the screw shaft member 46 axially using the rotational drive force transferred from the worm gear mechanism 5.

The second gear mechanism 6 is mounted in the base member 2. This second gear mechanism 6 includes a threaded hole 45 formed concentric to the center of the worm wheel 41 and the screw shaft member 46 screwed to the threaded hole 45. A bolt 47 is inserted from below into the center of the screw shaft member 46, and the threaded section of the bolt 47 projecting upward from the screw shaft member 46 is linked to a conversion member 50 of the conversion mechanism 7 so that the screw shaft member 46 and the conversion member 50 are linked in a fixed manner.

A collar member 48 is secured between the screw shaft member 46 and the head of the bolt 47. This collar member 48 has a diameter that is slightly less than that of a hole 25a of the cover 25. When the screw shaft member 46 is in a lowered state, the collar member 48 fits into the hole 25a, and the axial center of the screw shaft member 46 and the rotational center of the worm wheel 41 are placed in fixed positions.

Referring to Fig. 1, Fig. 2, Fig. 4, and Fig. 5, the conversion mechanism 7 changes the direction of the axial drive force transferred by the screw shaft member 46 and transfers this force to the pair of claw members 3, moving these left and right claw members 3 symmetrically. This conversion mechanism 7 includes the conversion member 50, a sloped engagement groove 51, and a pair of engagement sections 52.

The conversion member 50 is secured to the screw shaft member 46 by the bolt 47. This conversion member 50 is disposed within the vertical holes 13, 23 of the base member 2. The conversion member 50 is formed with a pair of T-

~~shaped sloped engagement groove 51 sloped in the direction of movement of the claw members 3. The pair of claw members 3 are formed with a pair of engagement sections 52 to slidably engage with the pair of sloped engagement grooves 51. Since the pair of engagement sections 52 engage with the pair of sloped engagement grooves 51, the conversion member 50 is prevented from rotating relative to the base member 2.~~

~~Referring to Fig. 7 and Fig. 8, the sloped engagement grooves 51 are sloped at approximately 70 degrees relative to the direction of motion of the claw members 3 (the horizontal direction) so that the lower the grooves are, the further they are from the axial center of the conversion member 50. The lower the conversion member 50 and the screw shaft member 46 go, the closer the pair of claw members 3 approach each other. Conversely, the higher the conversion member 50 and the screw shaft member 46 are, the further apart the pair of claw members 3 move. The legs 30 of the claw members 3 are formed with grease holes 3a. Grease in these grease holes 3a is fed between the sloped engagement grooves 51 and the engagement sections 52.~~

~~The operations and advantages of the two-claw chuck device 1 will be described.~~

~~Referring to Fig. 5, when chucking the workpiece W, the conversion member 50 is first moved upward so that there is adequate space between the pair of claw members 3. The workpiece W is then set onto the lowered section 12. Referring to Fig. 5, the conversion member 50 is moved to the uppermost position and the pair of claw members 3 are positioned so that they are separated by the maximum distance. However, when setting the workpiece W, the pair of claw members 3 does not necessarily need to be separated by the maximum distance.~~

Next, the end of a rotation tool such as a hexagonal wrench is engaged with the angular hole 4a of the input shaft member 4, and the input shaft member 4 is manually rotated with the rotation tool so that the pair of claw members 3 move toward each other. In the worm gear mechanism 5, the worm gear 40 rotates integrally with the input shaft member 4, and the worm wheel 41 meshed with the worm gear 40 rotates around the vertical axis.

In the second gear mechanism 6, the worm wheel 41 cannot move vertically when the worm wheel 41 is rotated, so the screw shaft member 46 screwed to the threaded hole 45 of the worm wheel 41 is driven downward, and the conversion member 50 secured to the screw shaft member 46 is driven downward as well.

In the conversion mechanism 7, when the conversion member 50 moves downward the operation of the sloped engagement grooves 51 of the conversion member 50 and the pair of engagement sections 52 cause the pair of claw members 3 to come closer, i.e., move toward the workpiece W. As shown in Fig. 4, the upper ends of the claw sections 31a of the pair of claw members 3 firmly chuck the workpiece W from the left and right. Machining is then performed on the workpiece W from this chucked state.

When removing the workpiece W, the rotation tool is used to rotate the input shaft member 4 in the opposite direction so that the claw members 3 move away from each other. The rotation causes the screw shaft member 46 to move upward via the worm gear mechanism 5 and the second gear mechanism 6. This causes the conversion member 50 secured to the screw shaft member 46 to move upward as well, driving the claw members 3 away from each other via the conversion mechanism 7. This releases the chucked state.

~~According to this chuck device 1, the worm gear mechanism 5 allows the rotational drive force input through the input shaft member 4 to be significantly increased (multiplied). Furthermore, the second gear mechanism 6 increases the rotational drive force transferred from the worm gear mechanism 5 and transfers it to the screw shaft member 46, driving the screw shaft member 46 in the axial direction. The axial drive force transferred by the screw shaft member 46 is redirected and increased by the conversion mechanism 7 and is transferred to the pair of claw members 3. This causes the claw members 3 to move symmetrically.~~

~~By providing the worm gear mechanism 5, the second gear mechanism 6, and the conversion mechanism 7 as described above, the drive force input through the input shaft member 4 can be increased in three stages and then transferred to the pair of claw members 3. As a result, simply applying a manual drive torque to the input shaft member 4 can firmly chuck the workpiece W. As a result, the usability of the chuck device can be improved and chucking operations on the workpiece W can be made more efficient, thus preventing reduced machining precision and damage to cutting tools.~~

~~A high ratio for increasing the drive force can be provided even without a very large ratio between the displacement stroke of the claw members 3 and the displacement strokes of the conversion member 50 and the screw shaft member 46.~~

~~Since the pair of claw members 3 is mounted on the upper surface of the base member 2 and the worm gear mechanism 5 and the second gear mechanism 6 are mounted inside the base member 2, the structure for increasing the drive force can be made compact. Also, entry of debris into the gear mechanisms 5, 6 can be prevented.~~

Since the claw members 3 are disposed facing each other and the legs 30 of the claw members 3 are slidably engaged with the shared engagement groove 11 formed in the base member 2, the shared engagement groove 11 can reliably guide and support the claw members 3 along their axis of motion. As a result, the workpiece W can be reliably chucked by being supported from either side by the claw members 3.

Next, examples of partial modifications to the above embodiment will be described.

1) The worm gear mechanism 5 can be omitted. If this is done, a rotation member is provided to substitute for the worm wheel 41, and the screw shaft member 46 is screwed into a threaded hole formed in this rotation member. Some sort of input member to input rotational drive force can be provided to rotate this rotation member, so that the screw shaft member 46 is driven axially.

2) Instead of the pair of claw members 3, a single claw member can be provided, and the workpiece can be chucked by supporting it between the claw member and a receiving section of the base member. Alternatively, three claw members can be provided, and the workpiece can be chucked using these three claw members. Changes in the number of claw members can be easily handled simply by changing the number and positions of the sloped engagement grooves formed on the conversion member in the conversion mechanism.

3) The input shaft member 4 can be rotated by an actuator such as a motor to apply rotational drive force through the input shaft member 4. Since rotation of the input shaft member 4 does not require a high drive force, the actuator can be compact, thus allowing the chuck device to be compact and reducing production costs.

4) An upper plate, used to chuck the workpiece by supporting it in cooperation with one of the claw members 3, can be secured to the lowered section 12 using bolts screwed into the lowered section 12. In this case, the other claw member 3 would not be used. However, by mounting the upper plate for workpieces having certain shapes, it would be possible to chuck workpieces smaller than the workpieces W chucked by the pair of claw members 3. Furthermore, after the upper plate is mounted, the upper plate can be cut into a shape corresponding to the shape of the workpiece using machine tools or the like. This provides reliable chucking for different workpiece shapes.

5) Various other modifications may be effected without departing from the spirit of the present invention. Also, the present invention can be used in chuck devices that secure the work piece to a rotating body of a machine tool or that secure tools to the principle axis.

Possible uses in industry

With the chuck device according to the present invention as described above, a light manual drive force can be applied to drive the claw members forcefully and provide firm chucking of workpieces or the like. Thus, a compact and high-performance chuck device for providing workpieces and tools is provided, and chucking operations can be made more efficient.

Abstract

A chuck device (1) includes: a worm gear mechanism (5) decelerating a rotational drive force applied through an input shaft member (4); a second gear mechanism (6) using the rotational drive force transferred from the worm gear

mechanism (5) to drive a screw shaft member (46) in the axial direction; and a conversion mechanism (7) redirecting the axial drive force transferred by the screw shaft member (46) to drive a pair of claw members (3) symmetrically. Thus, the rotational drive force applied through the input shaft member (4) is increased by the worm gear mechanism (5) and the second gear mechanism (6), and this rotational drive force is redirected by the conversion mechanism (7) and transferred to the claw members (3), allowing a workpiece or tool to be firmly chucked.